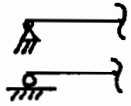
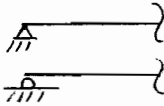
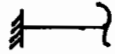
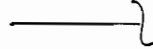
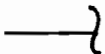


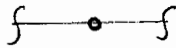
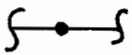
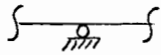
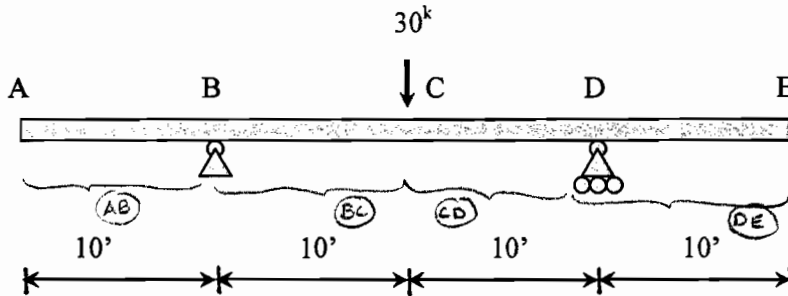


1. [15 pts] Fill in the right three columns of the following table. For the middle two columns, place an “equal” or “not equal” sign in the blanks. Then draw the type of conjugate support corresponding to the real support in the right-most column.

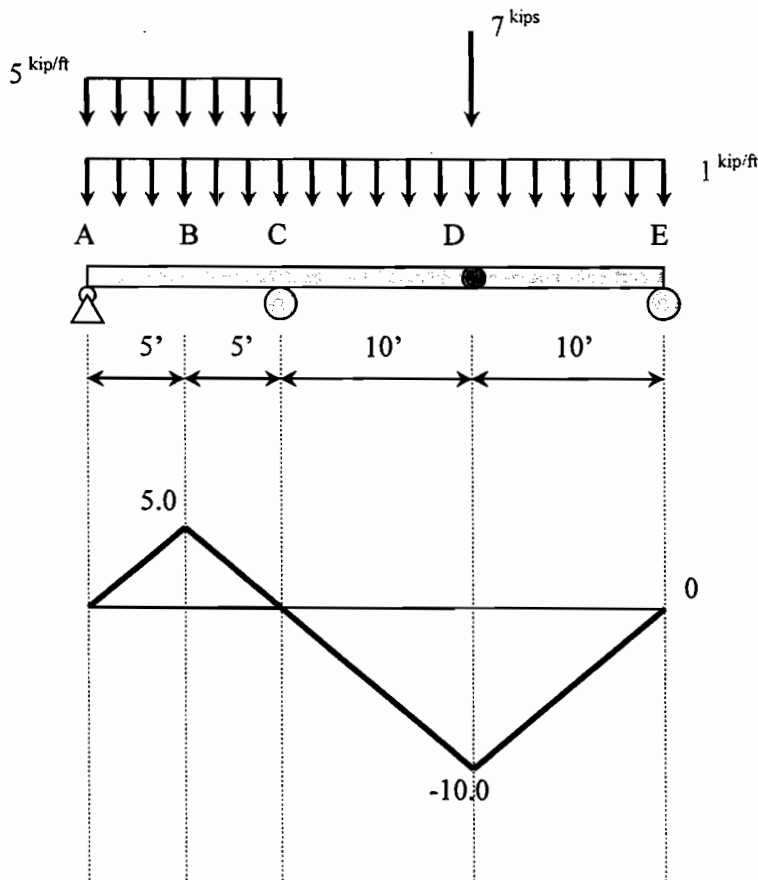
REAL BEAM		CONJUGATE BEAM	
Type of Support	Real Slope and Deflection	Conjugate Shear and Bending Moment	Type of Support
Simple end support 	$\theta \neq 0$ $\Delta = 0$	$V \neq 0$ $M = 0$	
Fixed support (encastre) 	$\theta = 0$ $\Delta = 0$	$V = 0$ $M = 0$	
Free end 	$\theta \neq 0$ $\Delta \neq 0$	$V \neq 0$ $M \neq 0$	
Simple interior support 	$\theta \neq 0$ $\Delta = 0$	$V \neq 0$ $M = 0$	
Internal hinge 	$\theta \neq 0$ $\Delta \neq 0$	$V \neq 0$ $M = 0$	

2. [5 pts] If you were using the method of direct integration to find the equations of the elastic curve for the entire length of the beam shown below (starting from the load function, not the moment function), how many constants of integration would you have to account for as part of your solution?



4 distinct regions -  
4 constants / region  
-----  
16 constants total

3. [10 pts] Given the following influence line for moment at B on the following beam, calculate the ~~absolute minimum~~ (positive/negative) moment at B generated by the loads shown.



$$M_B = 5 \text{ kip/ft} \left[ \frac{1}{2} \cdot 10 \cdot 5 \right] + 1 \text{ kip/ft} \left[ \left( \frac{1}{2} \cdot 10 \cdot 5 \right) - \left( \frac{1}{2} \cdot 20 \cdot 10 \right) \right] + 7 \text{ kips} [-10]$$

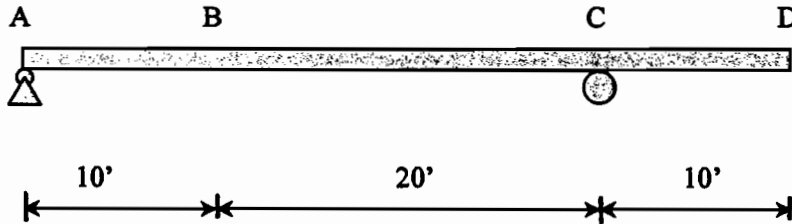
$$M_B = [125 \text{ k}'] - [75 \text{ k}'] - [70 \text{ k}']$$

$$M_B = -20 \text{ k}'$$

4. [35 pts] Given the following beam, draw the influence lines for:

- (10/35) Vertical reaction at A
- (10/35) Vertical reaction at C, and
- (15/35) Moment at point B.

You may use either the equilibrium method or Mueller-Breslau's Principle to complete the problem, but please clearly state which you have chosen.

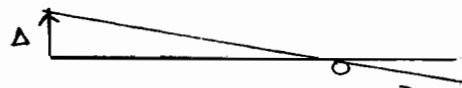


I have chosen to use Mueller-Breslau's Principle

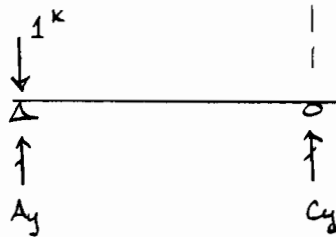
I.L. for Vertical Reaction at A,  $A_y$ .

Release the vertical rxn at A & allow to deflect:

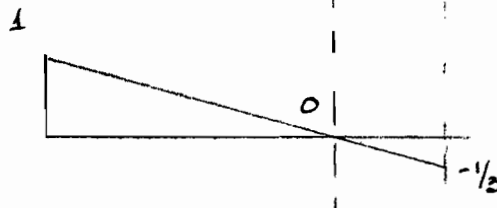
Apply a unit load at A & solve for  $A_y$  to determine the ordinate at A on the I.L.:



Shape of the I.L.



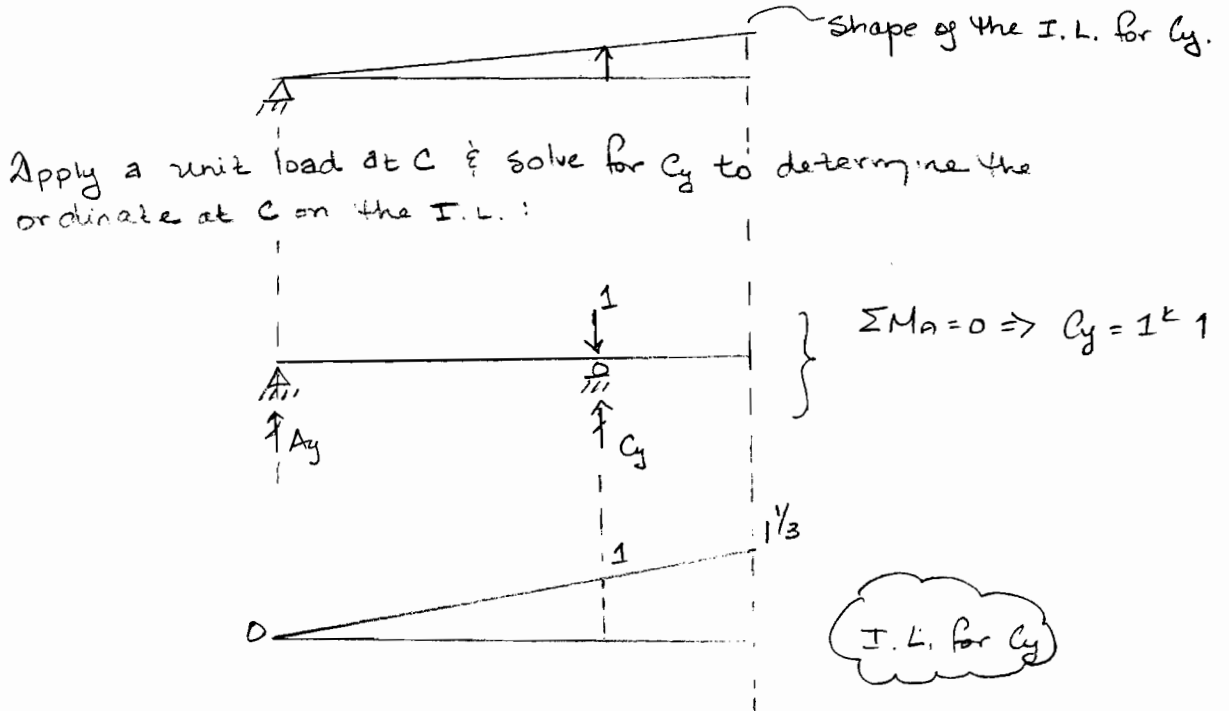
$$\sum M_c = 0 \Rightarrow A_y = 1^k \uparrow$$



I.L. for  $A_y$

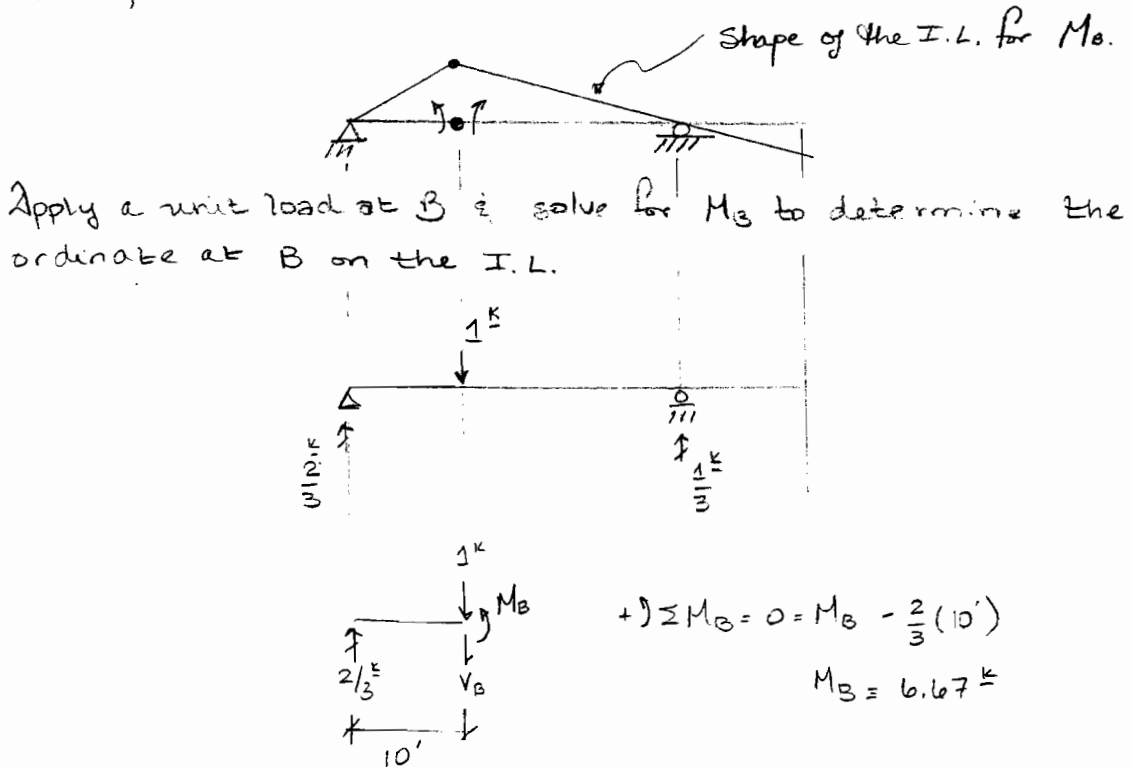
I.L. for Vertical Rxn @ C,  $C_y$ :

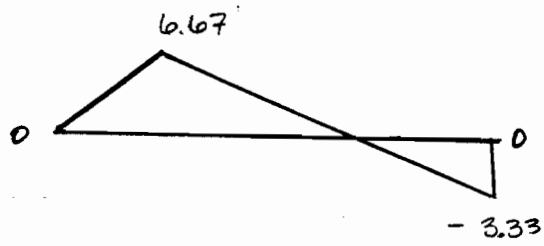
Release the vertical rxn @ C, & allow to deflect:



I.L. for Moment at B,  $M_B$ :

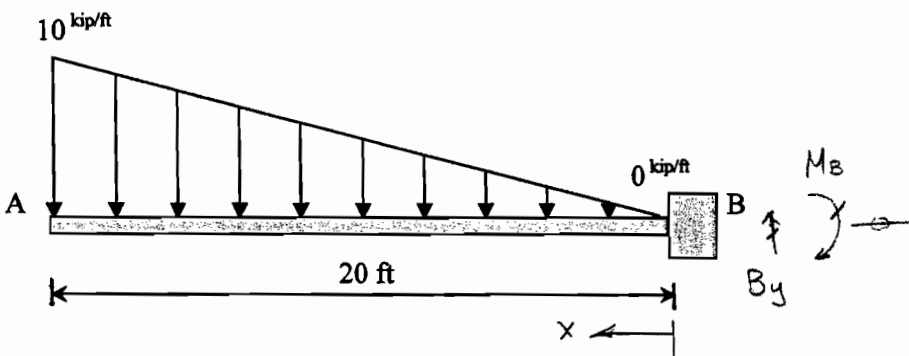
Release moment at B by inserting a hinge at B, and allow to deflect:





I.L. for  $M_B$

5. [35] Determine the maximum deflection of the following loaded beam under the given loads, using the method of direct integration. You may start with the load function or the moment function. Let  $E=29,000$  ksi and  $I=4200$  in<sup>4</sup>. Your answer should be given in inches.



Reactions:  $+\uparrow \sum F_y = 0 = -10 \text{ kip/ft} (20') (\frac{1}{2}) + B_y \Rightarrow \underline{B_y = 100 \text{ k} \uparrow}$

$+ \curvearrowright \sum M_B = 0 = M_B - 10 \text{ kip/ft} (20') (\frac{1}{2}) (\frac{2}{3} \cdot 20') \Rightarrow \underline{M_B = 1333.33 \text{ k-ft}}$

Taking coordinates from the fixity to the left,  $x \leftarrow 0$

(1)  $\boxed{w(x) = \frac{1}{2} x}$

(2)  $V(x) = \int -w(x) dx = \int -\frac{1}{2} x dx = -\frac{1}{4} x^2 + C_1$

$V(0) = 100 \text{ k} \therefore 100 \text{ k} = -\frac{1}{4} (0)^2 + C_1 \therefore C_1 = 100 \text{ k}$

$\boxed{V(x) = -\frac{1}{4} x^2 + 100}$

(3)  $M(x) = \int V(x) dx = \int -\frac{1}{4} x^2 + 100 dx = -\frac{1}{12} x^3 + 100x + C_2$

$M(0) = -1333.33 \text{ k-ft} \therefore -1333.33 = -\frac{1}{12} (0)^3 + 100(0) + C_2$

$\therefore C_2 = -1333.33$

$\boxed{M(x) = -\frac{1}{12} x^3 + 100x - 1333.33}$

(4)  $EI \theta(x) = \int M(x) dx = \int -\frac{1}{12} x^3 + 100x - 1333.33 dx$

$EI \theta(x) = -\frac{1}{48} x^4 + 50x^2 - 1333.33x + C_3$

$EI \theta(0) = 0 \therefore 0 = C_3$

$\boxed{EI \theta(x) = -\frac{1}{48} x^4 + 50x^2 - 1333.33x}$

$$(5) \quad EI y(x) = \int \theta(x) dx = \int -\frac{1}{48} x^4 + 50x^2 - 1333.33x \, dx$$

$$EI y(x) = -\frac{1}{240} x^5 + \frac{50}{3} x^3 - 666.67 x^2 + C_4$$

$$EI y(0) = 0 \quad \therefore 0 = C_4$$

$$EI y(x) = -\frac{1}{240} (x)^5 + \frac{50}{3} (x)^3 - 666.67 (x)^2$$

Max deflection will occur @ the free end,  
where  $x = 20'$ .

$$EI y(20') = -\frac{1}{240} (20)^5 + \frac{50}{3} (20)^3 - 666.67 (20)^2$$

$$EI y_{max} = -146,667 \text{ K}\cdot\text{ft}^3$$

$$y_{max} = \frac{-146,667 \text{ K}\cdot\text{ft}^3 (12'')^3}{29,000 \text{ ksi} (4200 \text{ in}^4)} = \boxed{-2.08'' \text{, neg} \therefore \downarrow}$$

(This is a big deflection,  
but the applied load was  
also very large for a  
cantilevered beam.)

